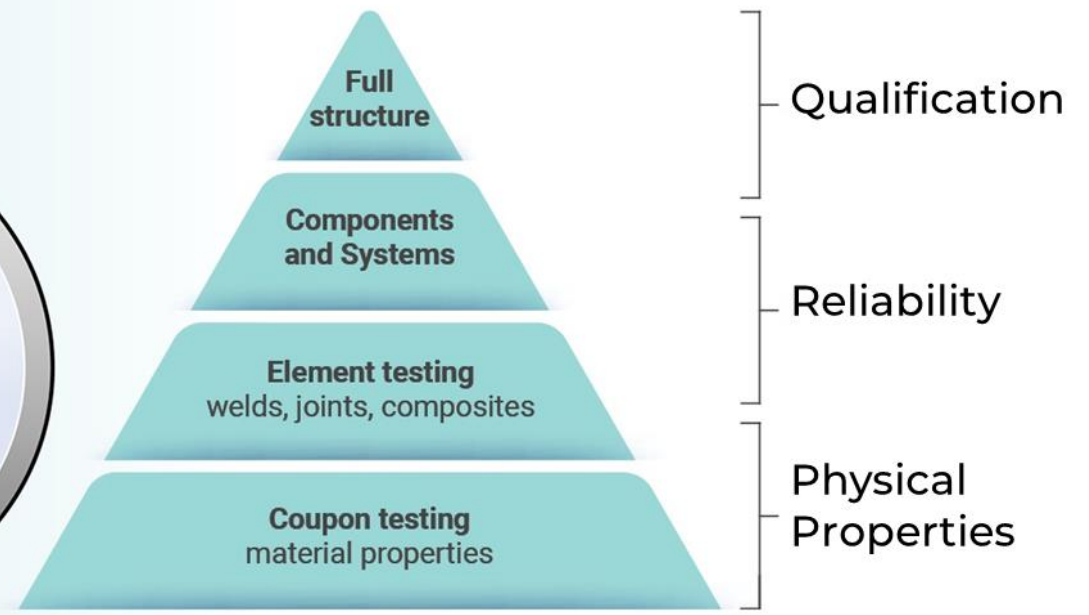
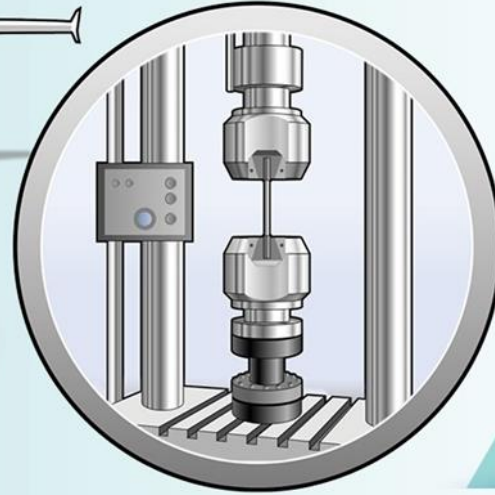
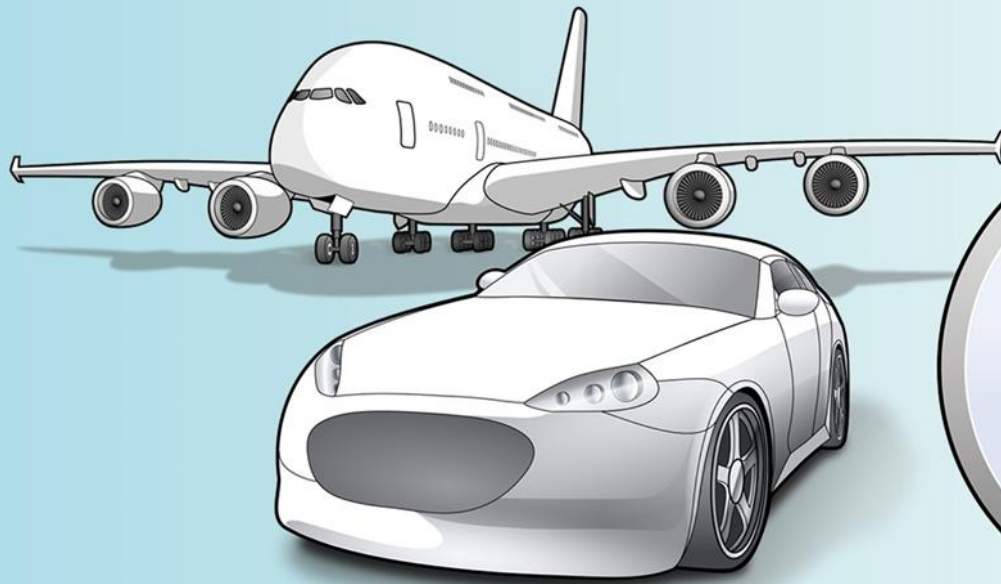
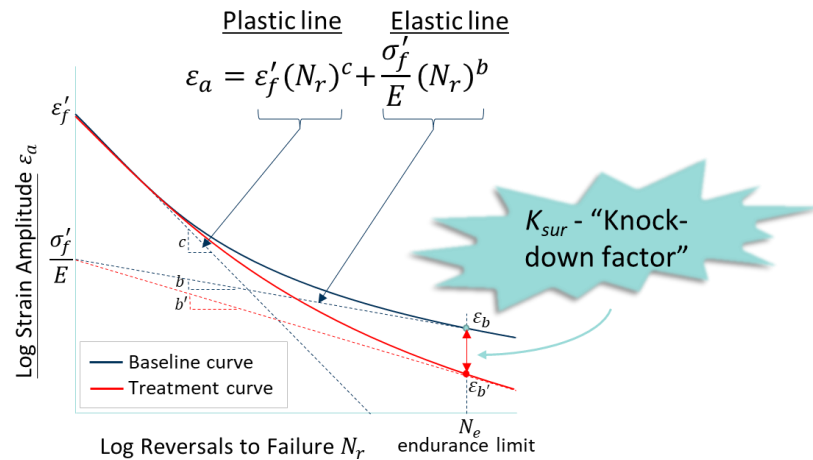




# 2023 HBK Technology Days

14 November 2023



- Fatigue methodology for combining surface treatment factors in strain-life fatigue

Dr. Andrew Halfpenny,  
Director of Technology – nCode Products, HBK

$$K_{sur} = \frac{\epsilon_{b'}}{\epsilon_b} = (N_e)^{b'-b}$$

# Fatigue methodology for combining surface treatment factors in strain-life fatigue

## Abstract

- The strain-life fatigue method is generally preferred for military aircraft landing gear fatigue life predictions because at the higher strain levels they experience, the stress-life fatigue method breaks down. The strain-life fatigue method requires strain controlled fatigue testing over a range of strain amplitudes and wide range of fatigue cycles to failure, followed by characterisation to fit parameters for the Ramberg-Osgood-Neuber cyclic plasticity model and Coffin-Manson-Basquin strain-life model. A minimum of 25 fatigue tests are recommended to characterise baseline material fatigue performance for a full range (10<sup>2</sup> to 10<sup>7</sup> cycles) equally biased low to high cycle strain-life curve.
- The  $K_{sur}$  method for surface treatment factors models the fatigue effect of a surface treatment with a single  $K_{sur}$  parameter to adjust the slope of a stress-life fatigue curve in the high cycle region. This method was originally derived for stress-life fatigue curves and assumes that surface effects are mostly confined to the high cycle fatigue region. It is reasoned that under high cycle fatigue loading, the applied loads are relatively low and localised fatigue initiation sites are dependent on the surface condition. Fatigue testing can be reduced to a minimum of 12 because there is only a single  $K_{sur}$  factor parameter to fit, and tests can be biased for results in the mid to high cycle fatigue region.
- This presents how strain-life fatigue parameters can be combined with  $K_{sur}$  surface treatment factors, including sample results and fatigue methodology validation.

# Fatigue methodology for combining surface treatment factors in strain-life fatigue

▲ About the presenter, Dr. Andrew Halfpenny, Director of Technology – nCode Products, HBK

- Dr. Halfpenny has a PhD in Mechanical Engineering from University College London (UCL) and a Master's in Civil and Structural Engineering. With over 25 years of experience in structural dynamics, vibration, fatigue and fracture, he has introduced many new technologies to the industry including: FE-based vibration fatigue analysis, crack growth simulation and accelerated vibration testing. He holds a European patent for the 'Damage monitoring tag' and developed the new vibration standard used for qualifying UK military helicopters.
- He has worked in consultancy with customers across the UK, Europe, Americas and the Far East, and has written publications on Fatigue, Digital Signal Processing and Structural Health Monitoring. He sits on the NAFEMS committee for Dynamic Testing and is a guest lecturer on structural dynamics with The University of Sheffield.

# Materials and Surface Coatings and Processes

2023 HBK TECHNOLOGY DAYS



Presenters:

Dr Michelle Hill, Head of Materials testing, HBK

Mr Ben Griffiths, Chief Engineer, Select Engineering Services

Dr Andrew Halfpenny, Director of Technology – nCode products, HBK

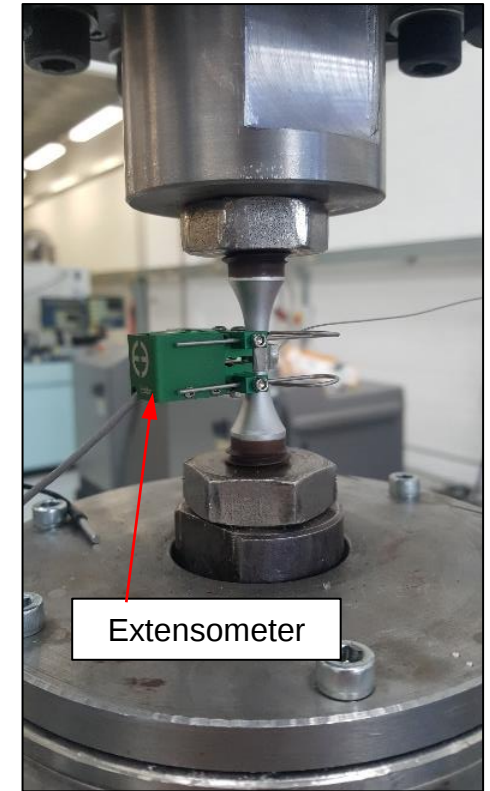
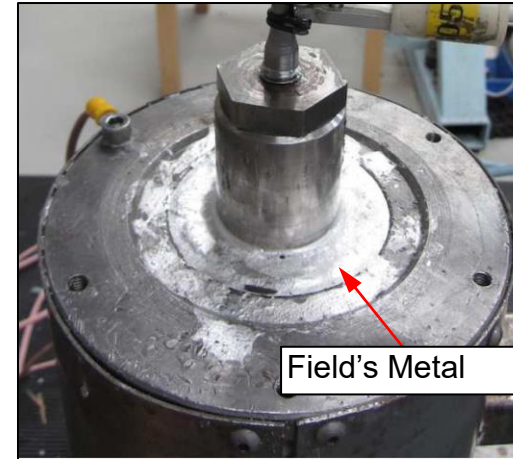


# Agenda

1. Materials and surface coatings and processes
  - Introduction
  - USAF Project Background and Significance
2. **Fatigue methodology for combining surface treatment factors in strain-life fatigue**
  - **Strain-Life Testing**
  - **Theory and Calculations**
3. Application of the Material Assessment and Predictive Analysis (MAPA) Tool for Fatigue Data Management
  - Tested Results
  - MAPA Data Manager
  - Ongoing Efforts

# Strain-life Testing

- Conducted at the AMCT Facility (HBK)
  - **Fully reversed** uniaxial strain – ASTM E606
  - **Strain-control** maintained for entire test
  - **Field's metal pot** for specimen alignment
- Specimen Fabrication
  - **Baseline** (ideal) specimens longitudinally polished
  - **Surface treated** specimens fabricated to USAF landing gear specs



Fatigue Test Setup

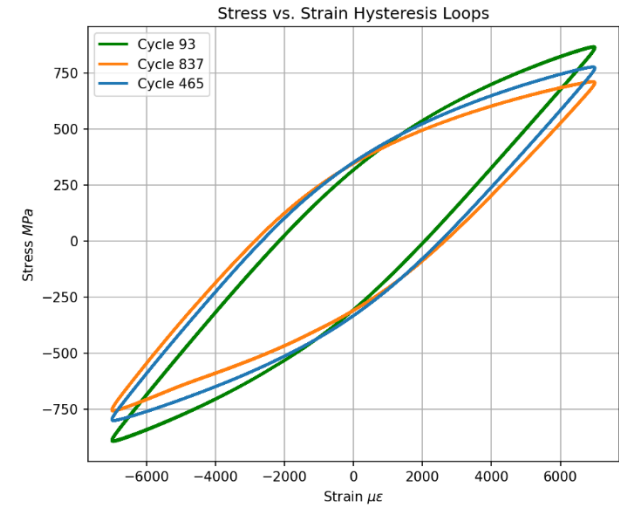
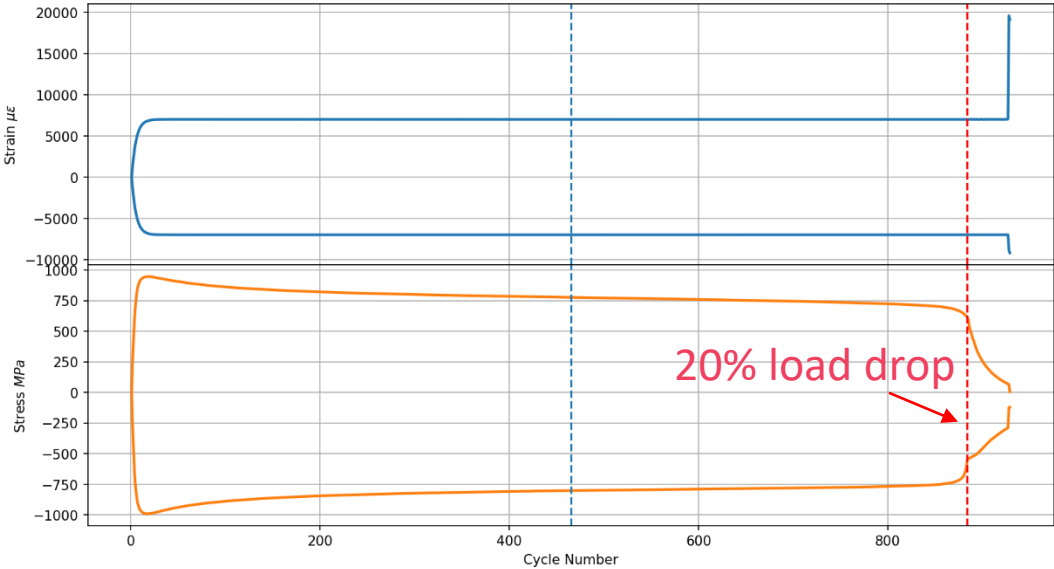
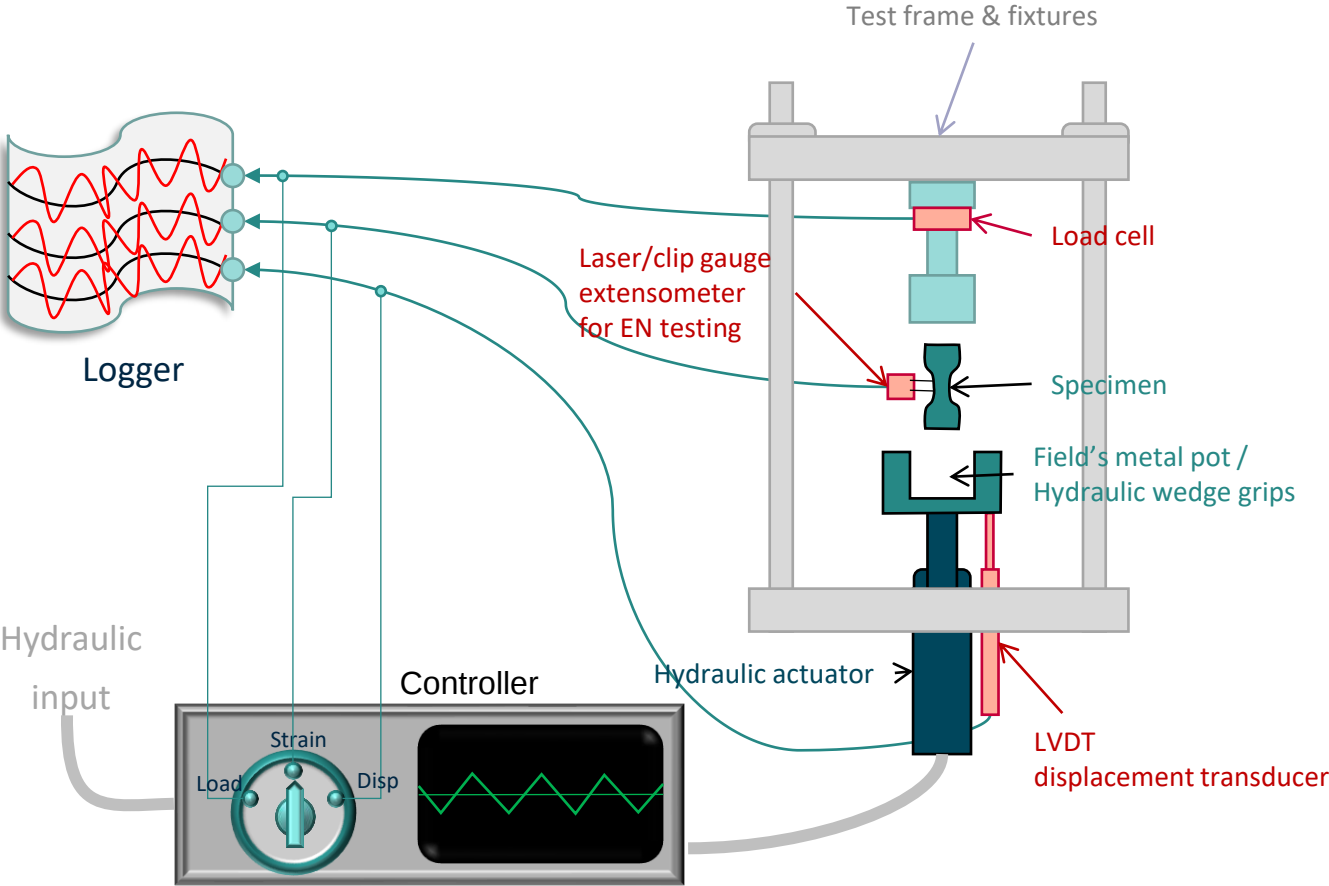


Longitudinal Polish

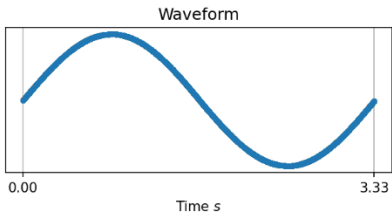


# Cyclic & Fatigue Parameters

Job reference: C12345      Material:      Sample Material  
Test number: 11      Customer: HBK AMCT  
Test date: 21/10/2020      Engineer: MC  
Software version: 3.0.1      Test rig: Emily (Tiab)



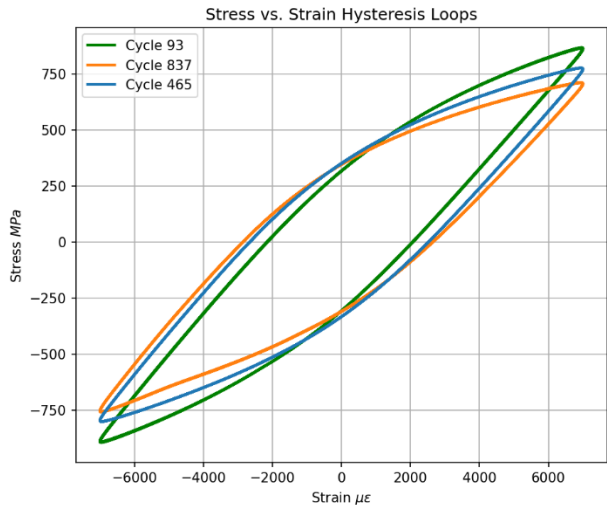
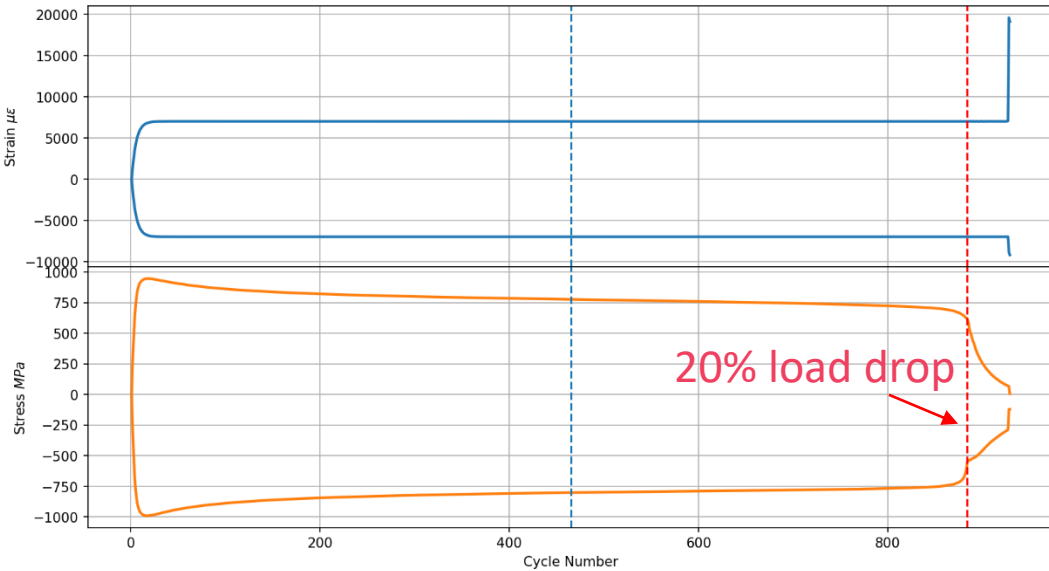
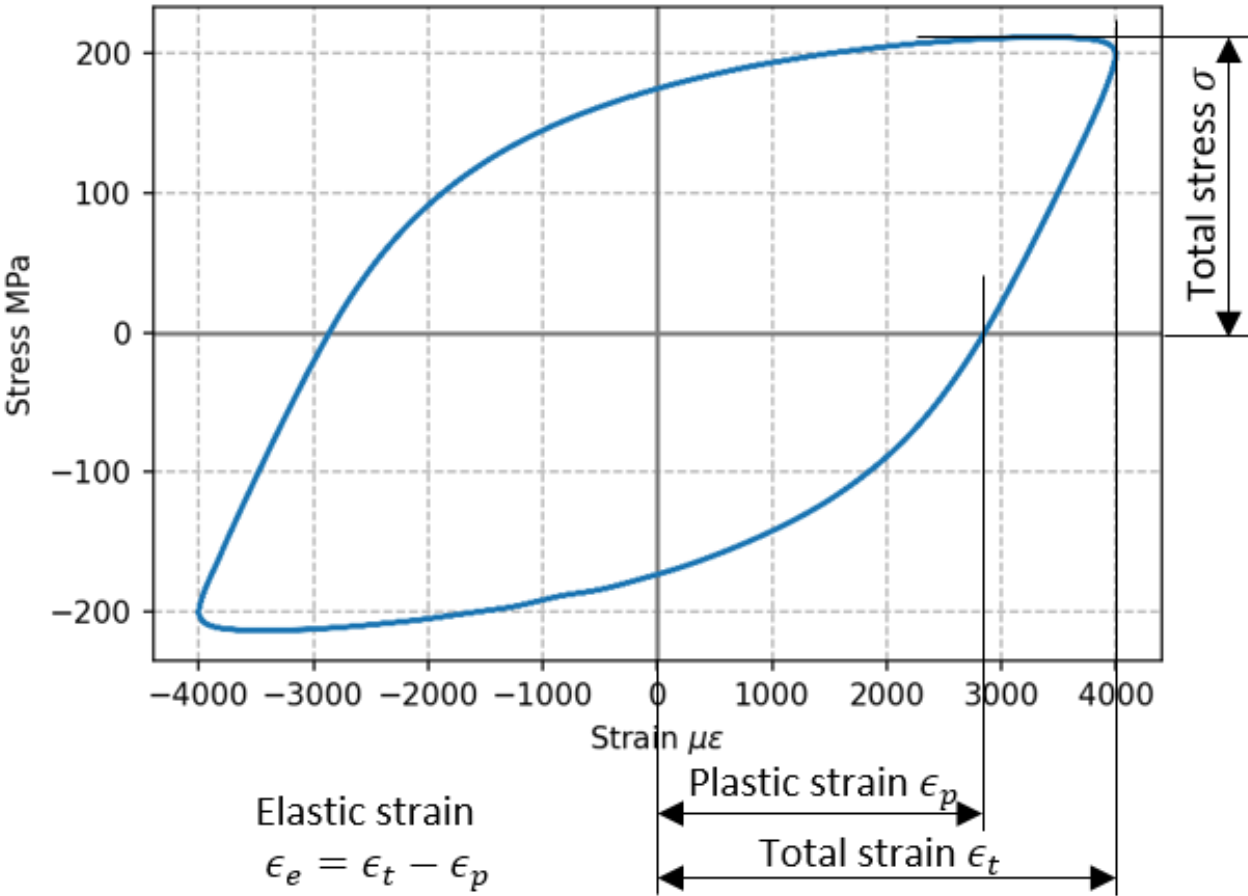
Cross sectional area: 16.100 mm<sup>2</sup>  
Test temperature: Ambient  
Test frequency: 0.3 Hz  
Stable loop: 465  
Cycles to separation: 929  
5% load drop: 740  
10% load drop: 858  
20% load drop: 884  
50% load drop: 893



| Cycle No. | Total Stress | Total Strain | Elastic Strain | Plastic Strain |
|-----------|--------------|--------------|----------------|----------------|
| 93        | 1758.5 MPa   | 14000        | 9780           | 4220           |
| 465       | 1577.1 MPa   | 13999        | 8817           | 5182           |
| 837       | 1468.4 MPa   | 14000        | 8496           | 5504           |

# Cyclic & Fatigue Parameters

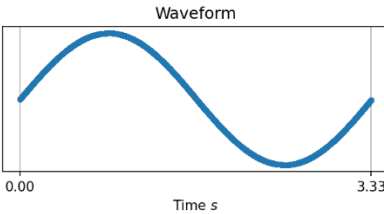
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Test temperature: Ambient  
Test frequency: 0.3 Hz  
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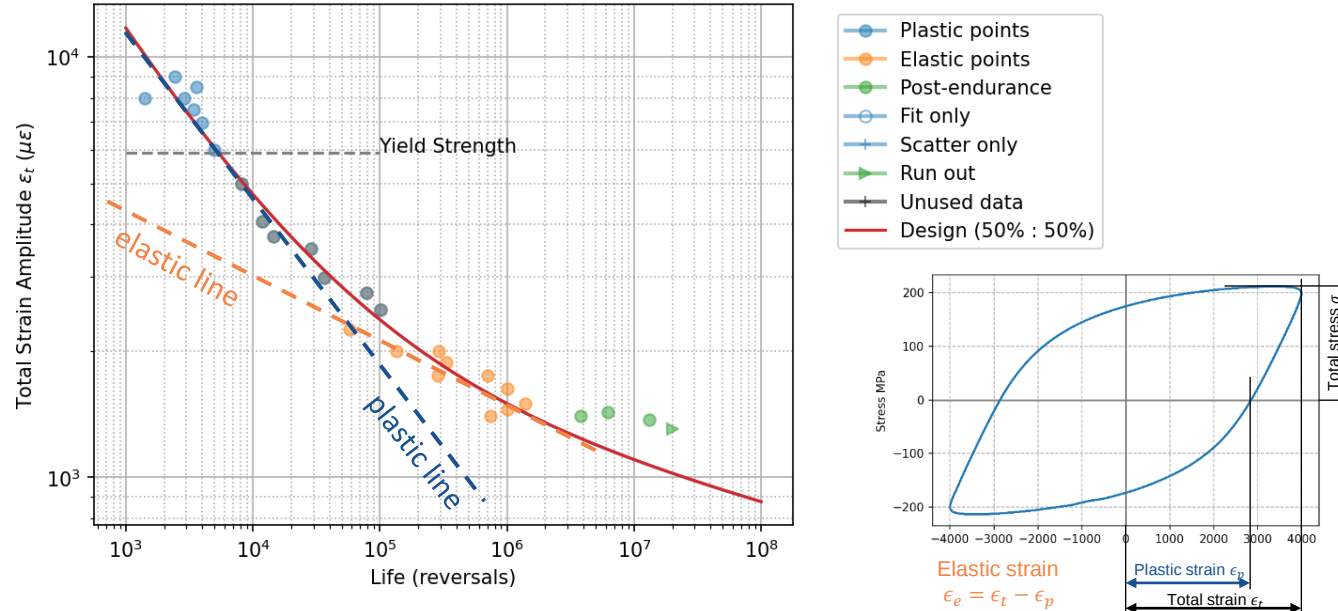
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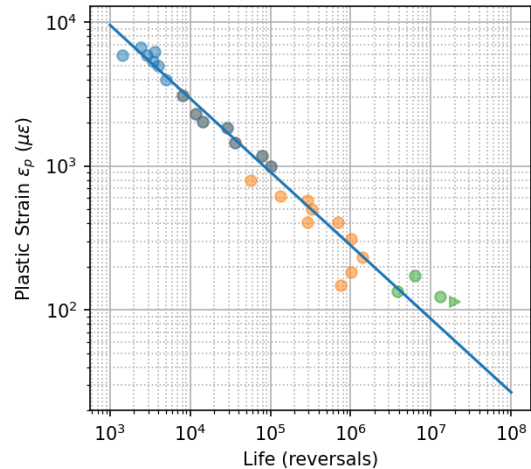
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# Cyclic & Fatigue Parameters – EN characterisation analysis

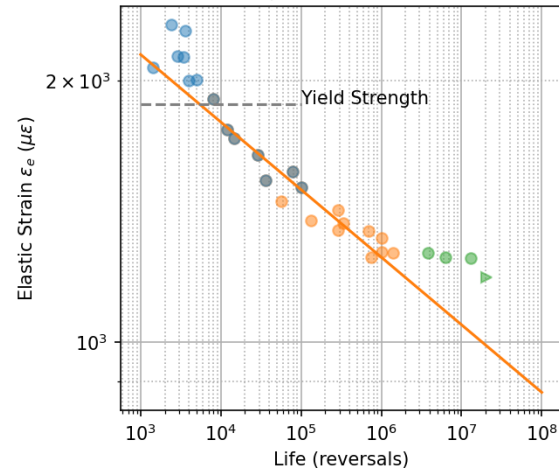
Strain-Life (EN) Curve



a) Plastic Curve



b) Elastic Curve



1. Plot Total strain amplitude  $\epsilon_t$  vs. reversals to failure  $N_r$  for the specified load-drop.

2. Fit **plastic line** using plastic strain  $\epsilon_p$

$$\epsilon_p = \epsilon_f' N_r^c$$

3. Fit **elastic line** using elastic strain

$$\epsilon_e = \frac{\sigma_f'}{E} N_r^b$$

4. Parameter re-optimisation

$$\epsilon_t = \frac{\sigma_f'}{E} N_r^b + \epsilon_f' N_r^c$$

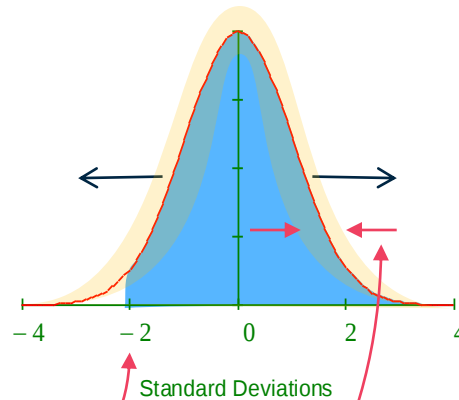
5. Outlier analysis

# Cyclic & Fatigue Parameters – Uncertainty Quantification

Deterministic design approach:

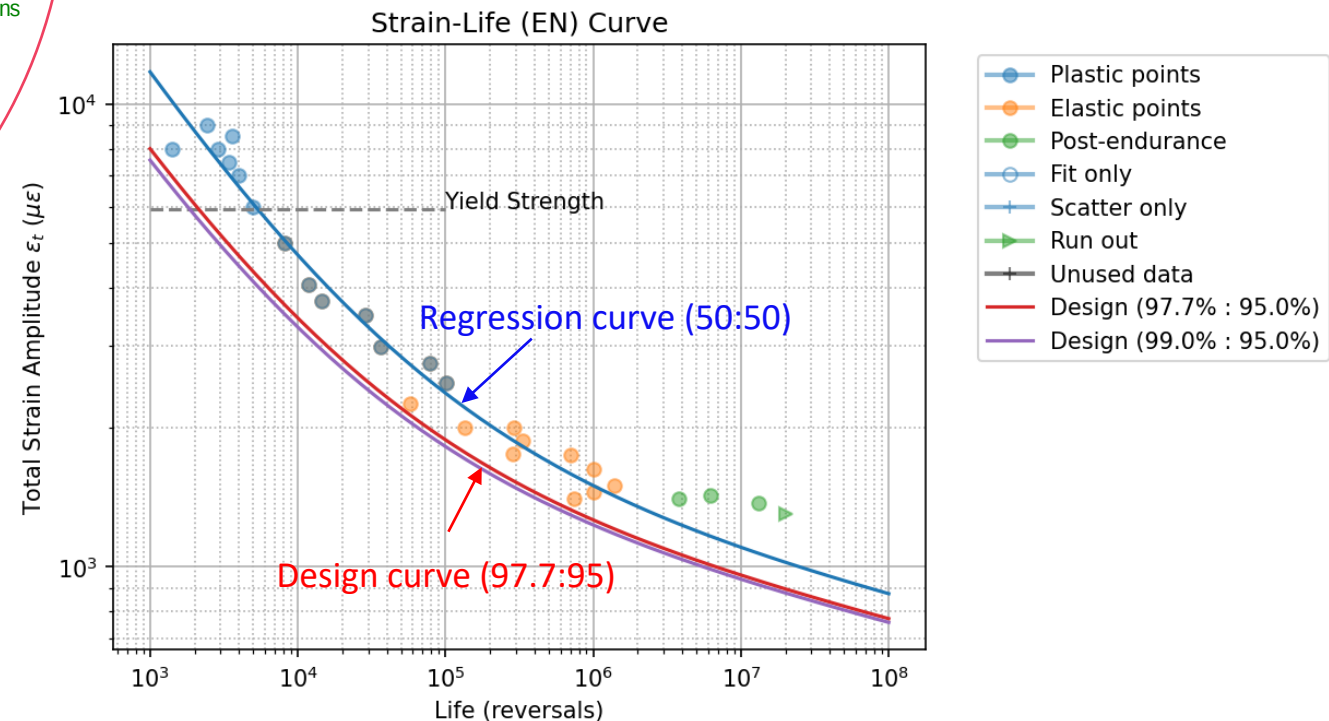
Design curve(reliability % : confidence %)

- Reliability statistic:  
(acceptable number of failures)
  - 50% (mean expected life)
  - 95% (5 failures in 100)
  - 97.7% (2 sigma)
  - 99% (1 failure in 100)
- Confidence statistic:  
(sensitivity to sample size)
  - 50% (mean expected life)
  - 90 – 95% (typical design curves)



$$\epsilon_t = \frac{\sigma'_f}{E} 10^{k_{Ue} \cdot SE_e N^b} + \epsilon'_f 10^{k_{Up} \cdot SE_p N^c}$$

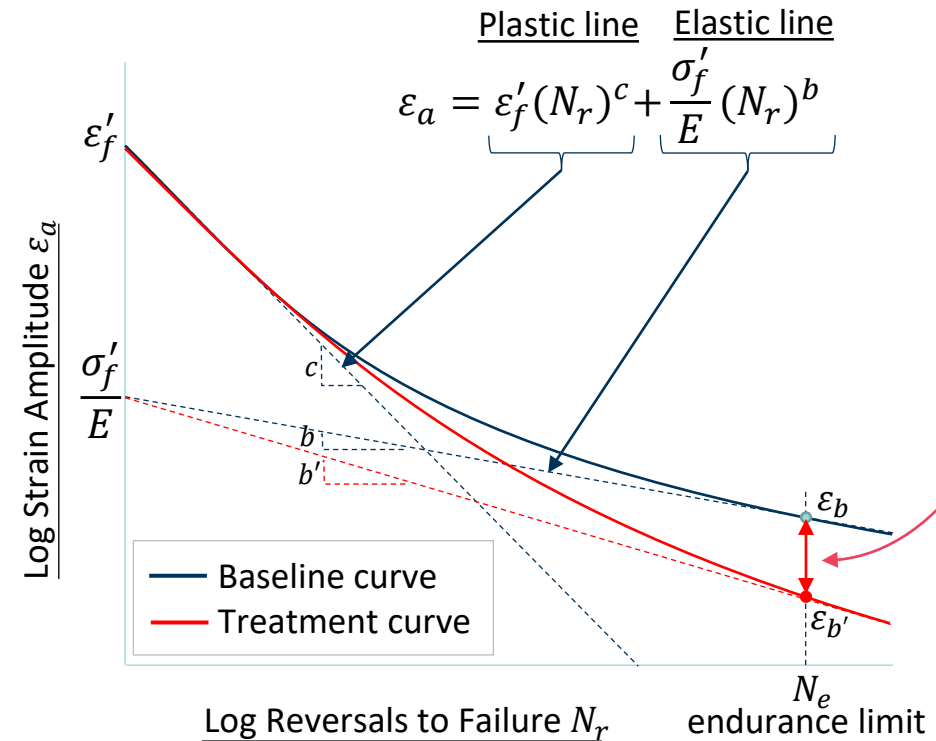
$$k_U = \frac{z_\rho + \frac{z_\gamma}{\sqrt{n}} \sqrt{1 - \frac{z_\gamma^2}{2(n-1)} + \frac{n z_\rho^2}{2(n-1)}}}{1 - \frac{z_\gamma^2}{2(n-1)}}$$



# Surface Treatment Factors – $k_{SUR}$ METHOD

“Knock-down factor”

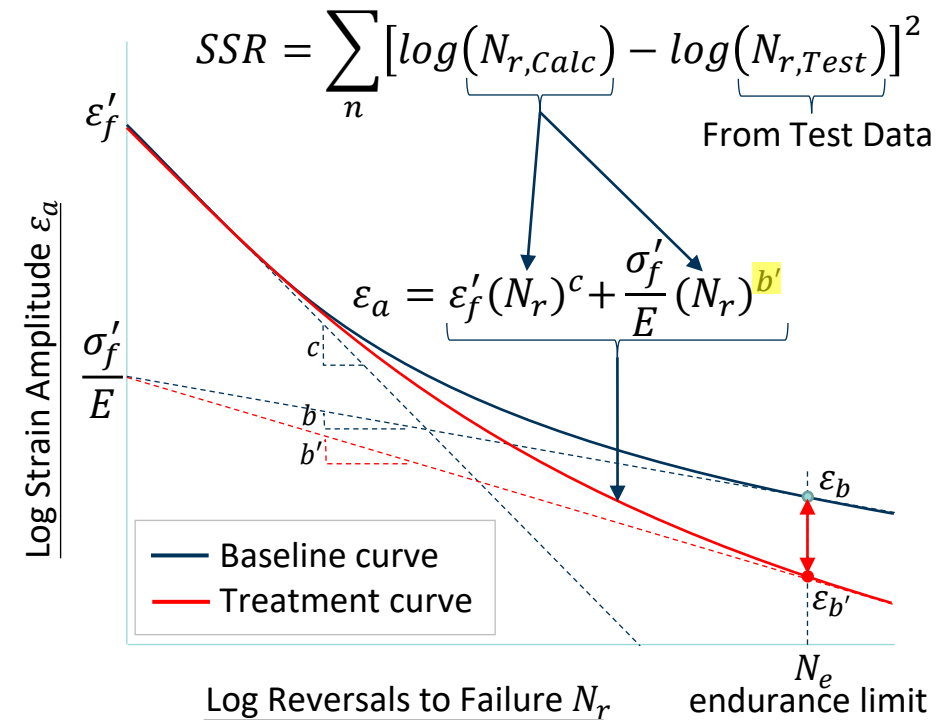
- Baseline curve (5 parameters)
  - 25 data points, fully reversed strain (R=-1)
  - High & low-cycle characterization
- Surface treatment test data
  - 12 data points, fully reversed strain (R=-1)
  - Focused on high-cycle fatigue
- Attributes of  $K_{sur}$ 
  - Single parameter treatment characterization
  - Similar statistical significance to 25-point curve
  - Compare different treatments on same material
  - Compare same treatments on different material
  - Assumes treatment effects greater at high cycles



$$K_{sur} = \frac{\varepsilon_{b'}}{\varepsilon_b} = (N_e)^{b'-b}$$

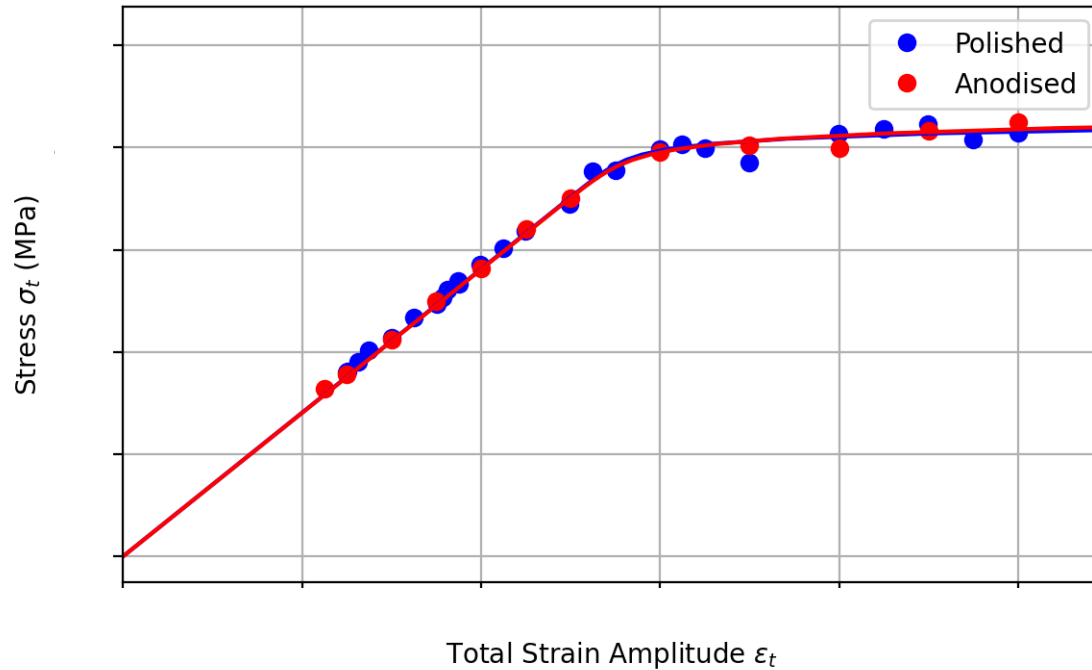
# $k_{SUR}$ Numerical calculations

- $K_{sur}$  is a function of  $b'$ ,  $b$ , and  $N_e$   $\longrightarrow K_{sur} = (N_e)^{b'-b}$ 
  - $b'$  - slope of elastic line for surface treatment curve
  - $b$  - slope of elastic line calculated with baseline (ideal) curve
  - $N_e$  - selected endurance limit
    - Does not affect surface treatment curve
- $b'$  calculated iteratively to minimize sum of squared residuals (SSR)
  - Cycles to failure considered dependent variable
  - Equation for SSR must be solved using numerical methods



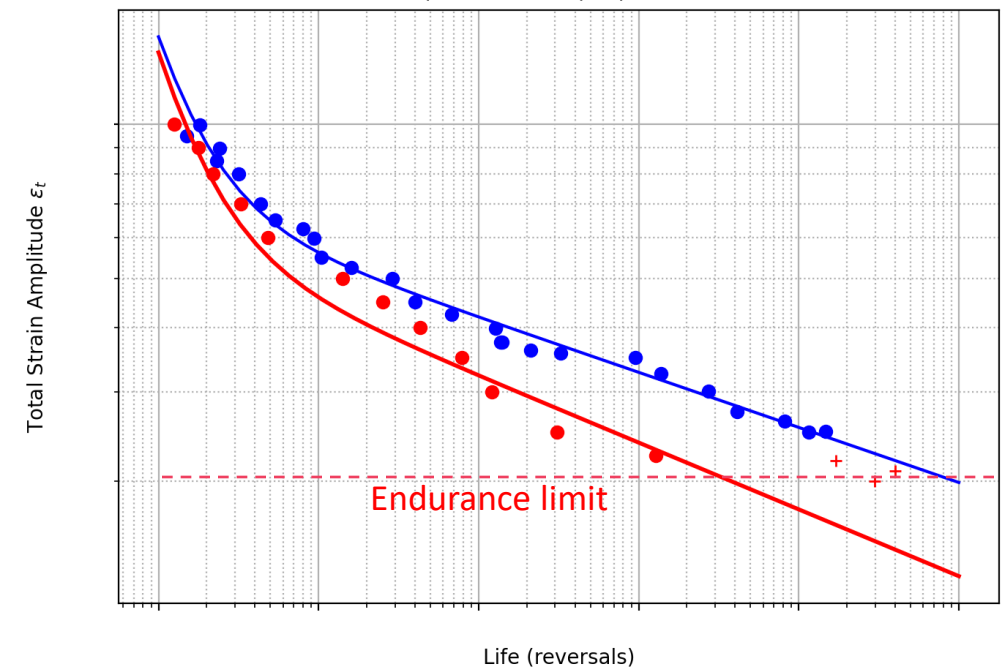
# E.g. 7075 – Anodised vs. Polished

Cyclic Stress vs. Strain Curve

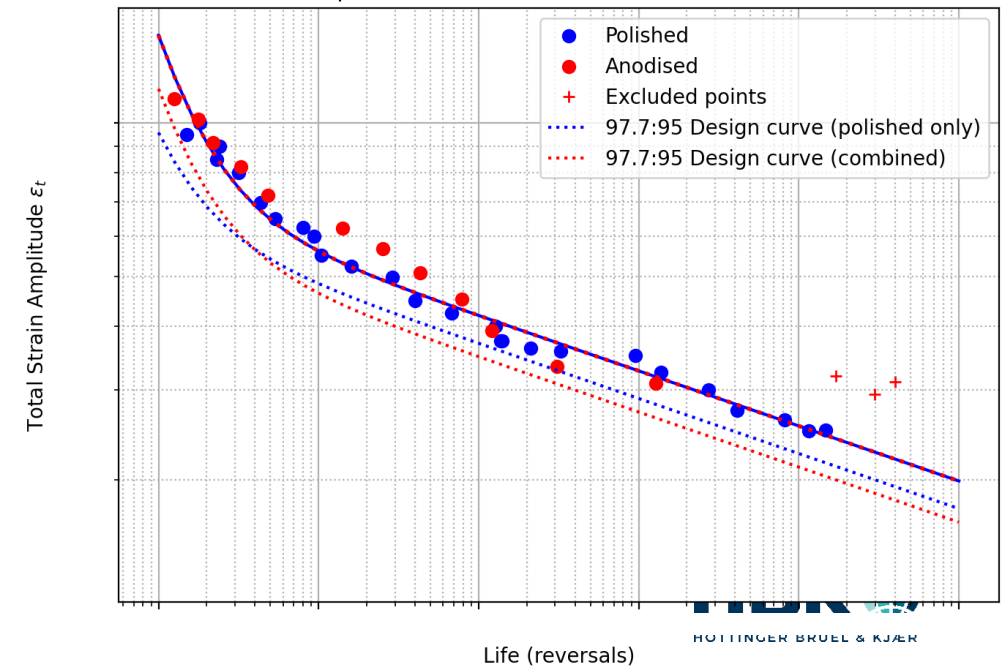


- ▲ No change in stiffness between baseline and treated
- ▲ Significant change in high-cycle fatigue life  $K_{sur} = 0.6801$
- ▲ Ksur method offers reasonable fit to material
- ▲ Endurance (fatigue strength) unmodified by treatment

a) Strain-Life (EN) Curve

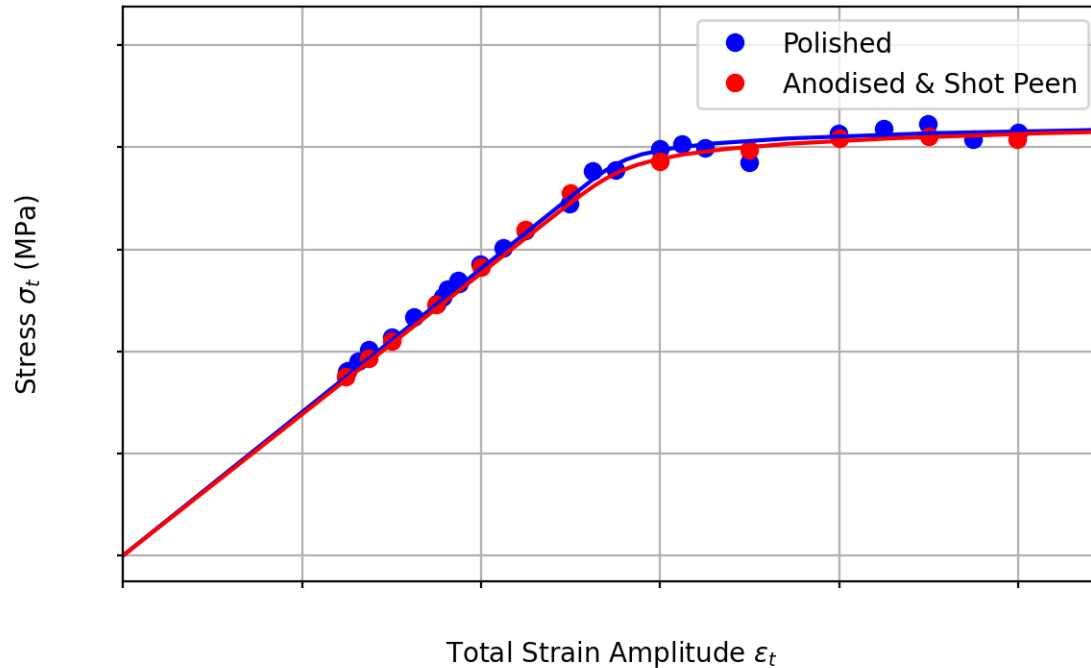


b) Collapsed Curve ( $K_f = 0.6801$  at  $N_e = 2.000e+07$ )



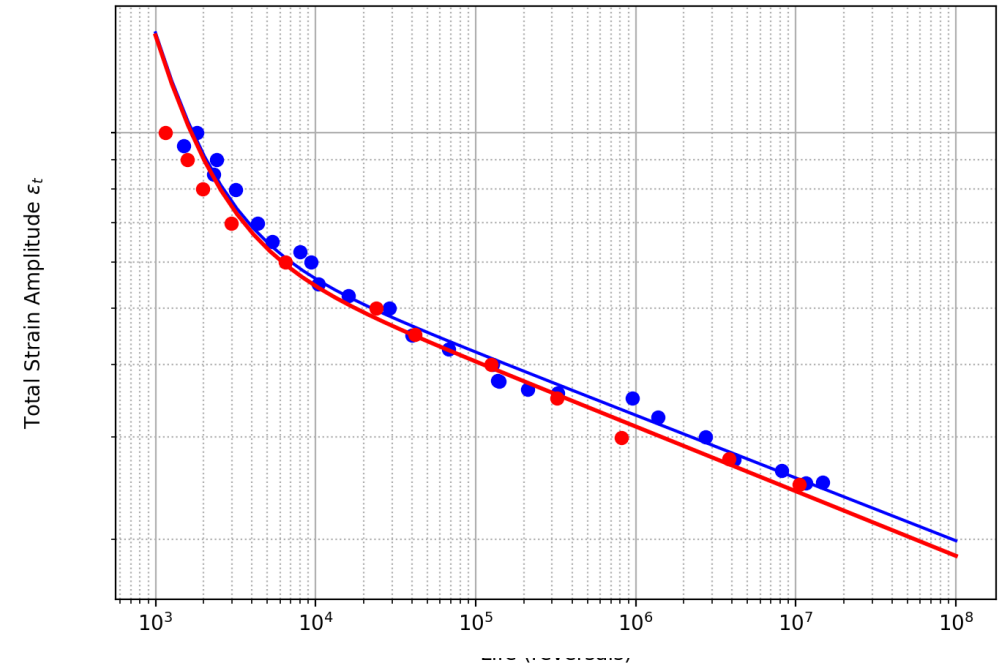
# E.g. 7075 – Shot peen & Anodised vs. Polished

Cyclic Stress vs. Strain Curve

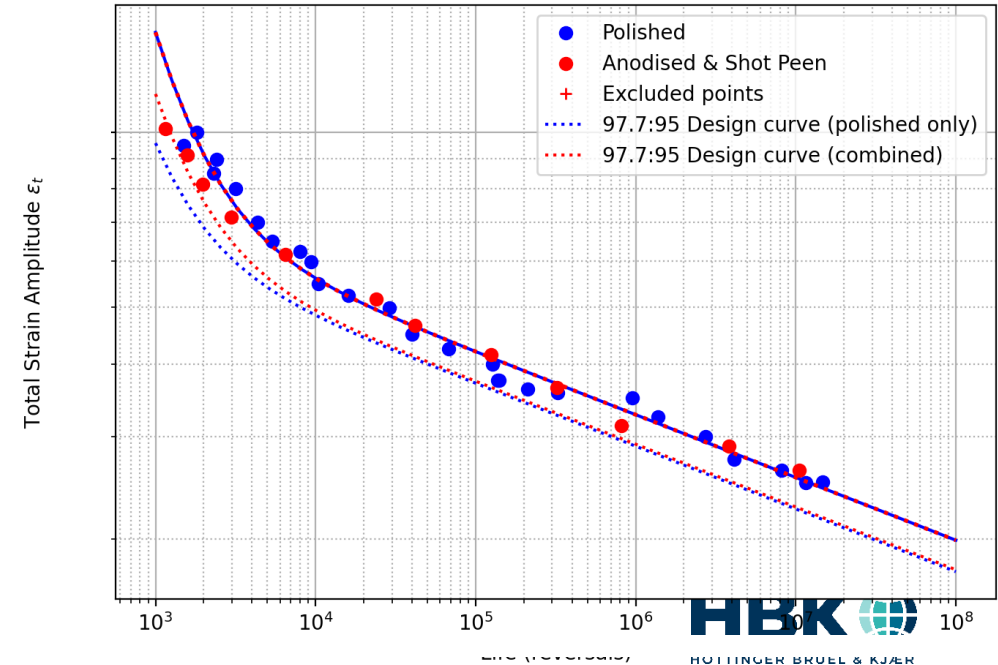


- ▲ No change in stiffness between baseline and treated
- ▲ Little change in high-cycle fatigue life  $K_{sur} = 0.9468$
- ▲ Shot peening effectively negates effect of anodising

a) Strain-Life (EN) Curve

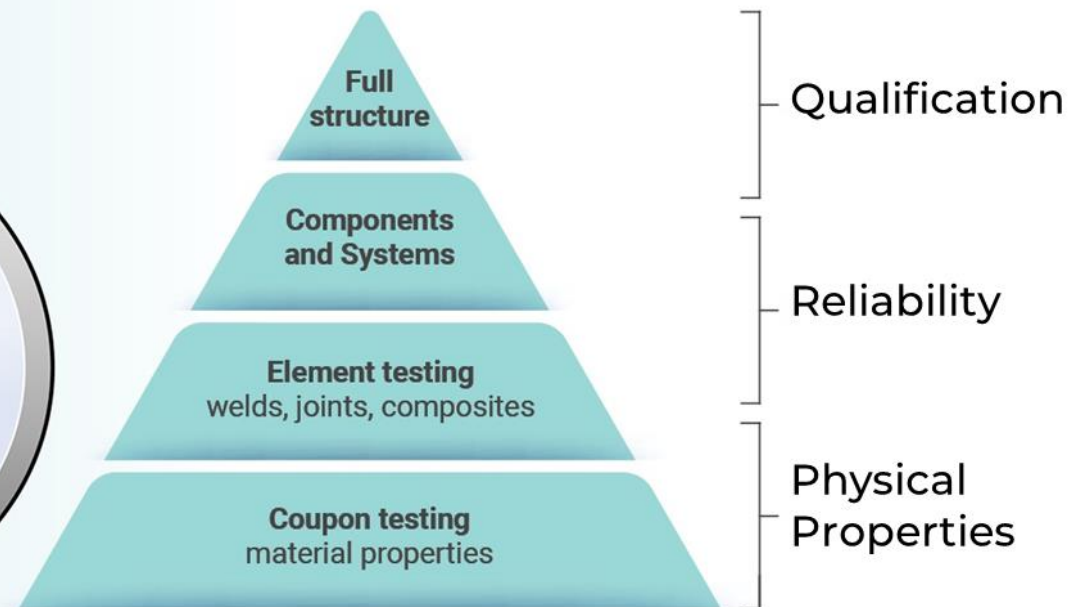
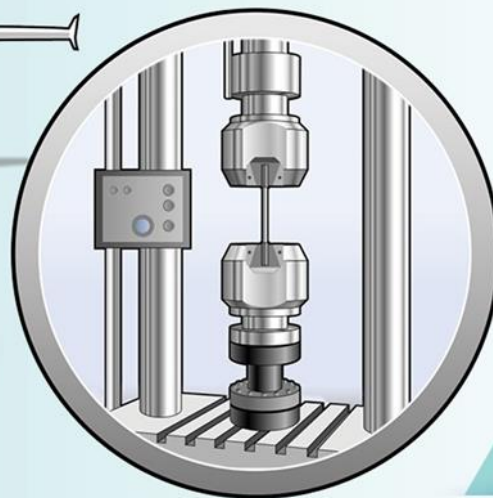
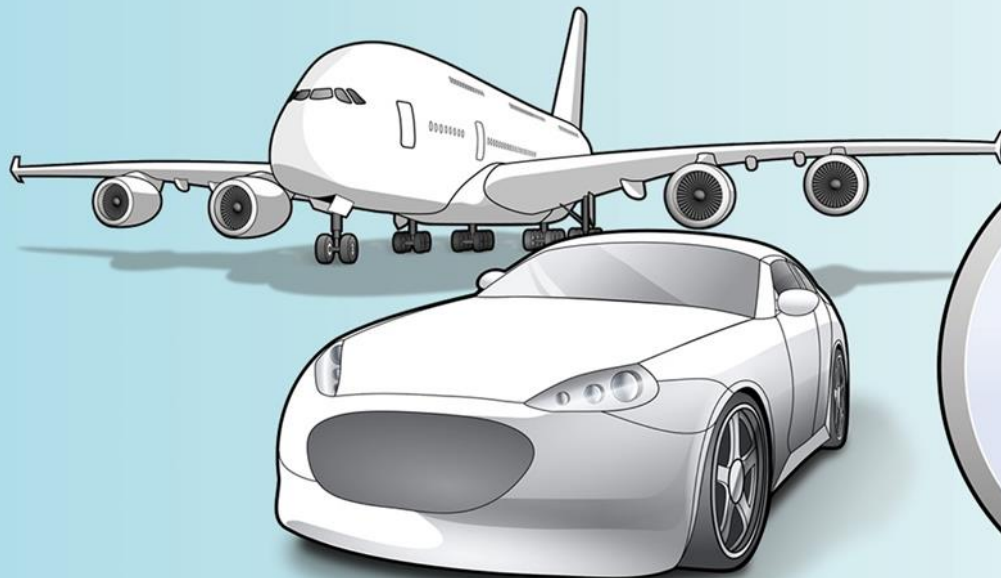


b) Collapsed Curve ( $K_f = 0.9468$  at  $N_e = 2.000e+07$ )



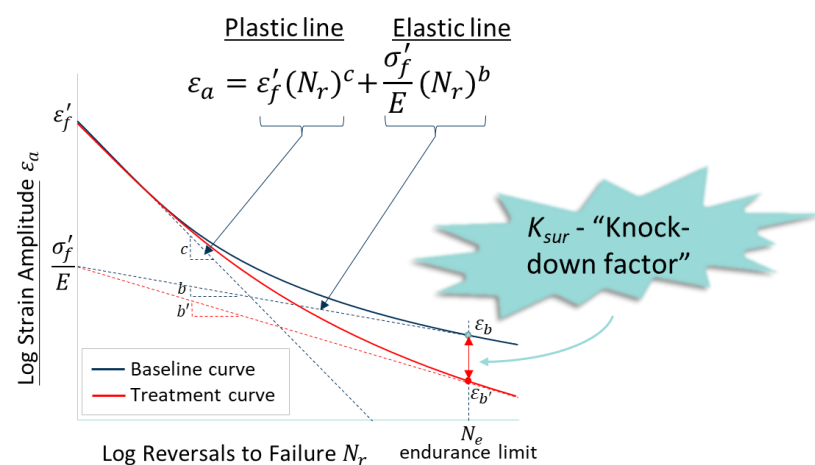
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